

## Contact Stress Calculation

### Input data

First radius body 1	$r_{11}$	5.000 mm
Second radius body 1	$r_{12}$	5.000 mm
First radius body 2	$r_{21}$	-5.2000 mm
Second radius body 2	$r_{22}$	100.00 mm
Effective length for cylinders	$l_{eff}$	0.0000 mm
Normal force	$F_n$	200.00 N
Youngs modulus body 1	$E_1$	210000 MPa
Youngs modulus body 2	$E_2$	210000 MPa
Poisson number body 1	$\nu_1$	0.3
Poisson number body 2	$\nu_2$	0.3
Angle between planes for radii	$\alpha$	0.0000 °

### Results

Major half axis of contact ellipsis	$a$	0.8172 mm
Minor half axis of contact ellipsis	$b$	0.0975 mm
Approach of both bodies	$\delta$	0.0036 mm
Contact stiffness	$R$	84259.6 N/mm
Hertzian stress	$\rho_H$	1198.6 MPa
Maximal shear stress body 1	$\tau_{Max_1}$	378.95 MPa
Maximal octahedral shear stress body 1	$\tau_{OctMax_1}$	324.94 MPa
Depth for max. shear stress body 1	$z(\tau_{Max_1})$	0.0747 mm
Equivalent stress body 1 (Tresca)	$\sigma_{eTresca_1}$	757.89 MPa
Equivalent stress body 1 (Mises)	$\sigma_{eMises_1}$	689.30 MPa
Maximal shear stress body 2	$\tau_{Max_2}$	378.95 MPa
Maximal octahedral shear stress body 2	$\tau_{OctMax_2}$	324.94 MPa
Depth for max. shear stress body 2	$z(\tau_{Max_2})$	0.0747 mm
Equivalent stress body 2 (Tresca)	$\sigma_{eTresca_2}$	757.89 MPa
Equivalent stress body 2 (Mises)	$\sigma_{eMises_2}$	689.30 MPa



